

* Laws:1 Major diameter:

$$D_{\text{major}} = R_f - R_i \quad [\text{Hand tools \& Abbe vertical}]$$

$$\rightarrow \text{Using B.Gs } (R_i = R_{\text{B.G.1}} + R_{\text{B.G.2}} + R_{\text{zero}} + R_{\text{anvils}})$$

$$D_{\text{major}} = R_2 - R_1 \quad [\text{Microscope \& projector}]$$

2 Minor diameter:

$$R = R_2 - (R_1 - R_{\text{B.G.}})$$

$$\psi = \tan^{-1} \left(\frac{\text{Lead std}}{\pi D_{\text{eff}}} \right) \rightarrow \text{Using B.Gs } (R_1 = R_{\text{B.G.1}} + R_{\text{zero}} + R_{\text{anvils}})$$

[Hand tools]

$$D_{\text{minor}} = R \cos \psi$$

$$D_{\text{minor}} = R_2 - R_1 \quad (\text{Consider helix angle}) \quad [\text{Microscope \& projector}]$$

3 Effective diameter:

$$D_e = R_2 - R_1 \quad [\text{Pitch dia. anvils}]$$

$$\rightarrow R_1 = R_{\text{anvils}} + R_{\text{zero}}$$

$$d_{\text{best}} = \frac{P_{\text{std}}}{2} \sec \theta_{\text{std}} \rightarrow \text{larger} \quad [3 \text{ wires \& Abbe vertical}]$$

$$d = \frac{R_2 - R_1}{2} \rightarrow R_{\text{zero}} \quad (\text{using wires only})$$

$$S = R_2' - R_1$$

$$D_e = S - d(1 + \cot \theta_{\text{std}}) + \frac{P_{\text{std}}}{2} \cot \theta_{\text{std}}$$

$$A = \frac{d}{T+d} = \frac{d}{S-d}$$

$$e = \frac{\cos \theta_{\text{std}} \cot \theta_{\text{std}}}{2\pi^2} \times \frac{L^2}{d} \times A^2 (1 + A \sin \theta + A^2 \sin^2 \theta)$$

$$c = 0.001 \frac{F^{2/3}}{D_c^{1/3}}$$

$$D_{\text{corrected}} = D_e + e + c$$

$D_e = R_2 - R_1$ (Consider helix angle) [Microscope & projector]

4 - Included angle measurement:

— Using values of eff. diameter and move dw_2 as well

$$\theta = \sin^{-1} \left[\frac{dw_2 - dw_1}{(Y_2 - dw_2) - (Y_1 - dw_1)} \right] \quad [\text{Hand tools & Abbe vertical}]$$

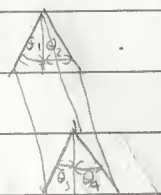
$$\theta_{\text{included}} = 2\theta$$

— Direct measurement [Microscope & projector]

5 - Flank angle measurement:

— Direct measurement and eliminate misalignment error

$$\theta_L = \frac{\theta_1 + \theta_3}{2}, \quad \theta_R = \frac{\theta_2 + \theta_4}{2}$$

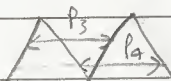
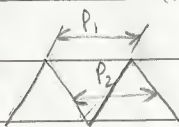


[Microscope & projector]

6 - Pitch measurement:

— $P = R_2 - R_1$ and eliminate misalignment error

$$P = \frac{\sum_{i=1}^4 P_i}{4}$$



7 - Virtual effective diameter:

$$\begin{aligned} \text{V.E.D.} = D_e + \underbrace{\Delta\theta}_{\text{flank}} + \Delta P &= s + d(1 + \cos \theta_{\text{std}}) + \frac{P_{\text{std}} \cot \theta_{\text{std}}}{2} \\ &+ 0.0131P(180\rho + 180)^2 + \frac{P}{2}(\cot \phi - \cot \theta) \\ &+ \Delta P_{\text{K\&A}} \cot \theta_{\text{std}} \quad \therefore = 0 \end{aligned}$$

[From std valves]

*Problems:1 Metric thread

$$D_e = 21.905 \text{ mm}$$

$$P = 3 \text{ mm}$$

$$\Delta P_{\max} = 0.025 \text{ mm}$$

$$\theta_R = 5'$$

$$\theta_L = 6'$$

An external thread of ISO metric form has the following dimensions and errors: Simple effective diameter = 21.905 mm, Pitch = 3.000 mm, Max. Pitch error = 0.025 mm, Right hand flank error = 5 min of an arc and the left hand flank error = 6 min of an arc. Determine the virtual effective diameter of the given thread.

$$\text{Error in flank angle of metric thread} = 0.0131 P_{\text{std}} (\theta_R + \theta_L)$$

$$\Delta \theta = 0.0131 \times 3 (5 + 6) \times \frac{1}{60} = 0.007 \text{ mm}$$

$$\text{Error in pitch} = \Delta P_{\max} \cot \theta_{\text{std}} = 0.025 \cot 30^\circ = 0.043 \text{ mm}$$

$$V.E.D. = D_e + \Delta \theta + \Delta P = 21.905 + 0.007 + 0.043 = 21.955 \text{ mm}$$

2 P = 5 mm, metric thread

$$R_t = R_{\text{cyl}} - D_{\text{cyl}} = 25.0033 - 15.0030 \\ = 10.0003 \text{ mm}$$

$$R_{\text{avg}} = \frac{\sum_{i=1}^3 R_i}{3} = \frac{13.2007 + 13.2002 + 13.2}{3} \\ = 13.2003 \text{ mm}$$

$$2d = R_{\text{avg}} - R_t = 13.2003 - 10.0003 \\ = 3.2 \text{ mm}$$

$$Pd = \frac{3.2}{2} = 1.6$$

$$R_{\text{avg}_5} = \frac{64.0272 + 64.027 + 64.0275}{3} = 64.0272 \text{ mm}$$

$$S = R_{\text{avg}_5} - R_t = 64.0272 - 10.0003 = 54.0269 \text{ mm}$$

$$A = \frac{d}{S - d} = \frac{1.6}{54.0269 - 1.6} = 0.0305$$

$$e = \frac{\cos 30^\circ}{2\pi^2 \tan 30^\circ} \times \frac{5^2}{1.6} \times (0.0305)^2 \left[1 + 0.0305 \sin 30^\circ + (0.0305 \sin 30^\circ)^2 \right] \\ e = 0.0018 \text{ mm}$$

In measuring the pitch diameter of a 52 mm metric plug screw gauge the average of the readings taken were:

- Reading over standard cylinder = 25.0033 mm
- Diameter of standard cylinder = 15.0030 mm
- Readings over each of the three wires = 13.2007, 13.2002 and 13.2000 mm
- Readings over thread with the three wires in three different places = 64.0272, 64.0270 and 64.0275 mm.

*Calculate the corrections for rake and compression errors, assuming a measuring force of 2.45 N.

*Calculate the effective diameter, taking into consideration the calculated rake and compression errors.

Subject. _____

Date. _____

26

$$D_e = S - d(1 + \operatorname{cosec} \theta) + \frac{P \sin \theta}{2} \cot \theta + \theta_{std}$$

$$= 54.0269 - 1.6(1 + \operatorname{cosec} 30^\circ) + \frac{5}{2} \cot 30^\circ = 53.5570 \text{ mm}$$

$$C = 0.001 \frac{F^{2/3}}{D_e^{1/3}} = 0.001 \frac{(2.45)^{2/3}}{(53.557 + 0.0018)^{1/3}} = 0.00048 \text{ mm}$$

$$D_{corrected} = D_e - e + C = 53.557 - 0.0018 + 0.00048 = 53.556 \text{ mm}$$

3

Metric thread

$$P = 1.5 \text{ mm}$$

$$S = 17.019 \text{ mm}$$

$$d = 1.35 \text{ mm}$$

$$\theta_R = 29.5^\circ$$

$$\theta_L = 30.5^\circ$$

$$P = 1.502 \quad 1.496 \quad 1.504 \quad 1.501 \quad 1.5 \quad 1.503 \quad 1.495 \quad 1.502 \text{ mm}$$

$$D_{ext} = 14.26 \pm 0.035 \text{ mm}$$

Calculate the virtual effective diameter of M16*1.5 screw if the reading over three wires placed over the thread is 17.019 mm. The wires having 1.35 mm diameter, the flank angles of the thread were 29.5° and 30.5° , and the pitch of the thread when checked along the teeth were:

1.502	1.496	1.504	1.501	1.500	1.503	1.495	1.502	mm
-------	-------	-------	-------	-------	-------	-------	-------	----

If this screw is to fit in a nut having 14.260 ± 0.035 mm pitch diameter, check whether the screw will fit fully, partially or will not fit at all when engaged with the nut.

$$P = 1.502 \quad 1.496 \quad 1.504 \quad 1.501 \quad 1.5 \quad 1.503 \quad 1.495 \quad 1.502 \text{ mm}$$

$$S \theta_R = \theta_R - \theta_{std} = 29.5^\circ - 30^\circ = -0.5^\circ$$

$$S \theta_L = \theta_L - \theta_{std} = 30.5^\circ - 30^\circ = 0.5^\circ$$

$$\Delta \theta = 0.0131 P \sin (\theta_R + \theta_L) = 0.0131 \times 1.5 (0.5 + 0.5) = 0.020 \text{ mm}$$

$$\Delta P_i = P_i - P_{std}$$

$$\Delta P = +0.002 \quad -0.004 \quad +0.004 \quad +0.001 \quad 0 \quad -0.003 \quad -0.005 \quad +0.002 \text{ mm}$$

$$\Delta P_{max} = 0.004 - (-0.005) = 0.009 \text{ mm}$$

$$\Delta P - \Delta P_{max} \cot \theta_{std} = \frac{0.009}{\tan 30^\circ} = 0.016 \text{ mm}$$

$$V.E.D. = D_e + \Delta \theta + \Delta P = S - d(1 + \operatorname{cosec} \theta) + \frac{P \sin \theta}{2} \cot \theta + \Delta \theta + \Delta P$$

$$= 17.019 - 1.35(1 + \frac{1}{\sin 30^\circ}) + \frac{1.5}{2 \tan 30^\circ} + 0.02 + 0.016$$

$$V.E.D. = 14.304 \text{ mm}$$

$$V.E.D_{min} = 14.225$$

$$V.E.D_{max} = 14.295 \text{ mm}$$

V.E.D. > V.E.D._{max} → Nut fits partially with 2nd thread

K&A
ANDLOSVA

$$14.304 - 0.016 = 14.288 \text{ (A)}$$

$$14.288 + 0.002 = 14.29 \text{ (B)} \checkmark$$

$$14.29 + 0.004 = 14.294 \text{ (C)} \checkmark$$

$$14.294 + 0.004 = 14.298 \text{ (D)} \times$$

4

Metric thread

$$P_{std} = 2 \text{ mm}, D_e = D_o - 0.956 \times 2 P = 12.536 \text{ mm}$$

$$d = 1.35 \text{ mm}$$

$$S.V. = 0.01 \text{ mm}$$

$$R_{range} = 25 \text{ mm}$$

$$R_{zero} = 0.05 \text{ mm}$$

$$All. D_e = 0.00 \pm 0.21 \text{ mm}$$

Three wire set of nominal size 1.35 mm was used to inspect the pitch diameter of M16*2 screw plug. The used external micrometer has a scale value of 0.01 mm, measuring range 25mm, and zero reading of 0.05mm. Knowing that the maximum and minimum permissible allowance of the pitch diameter is +0.21 mm and -0.00 mm, determine the extreme readings of the micrometer such that the readings correspond to the accepted screw lies-in-between.

$$R = S + \text{zero error} + \text{allowance}$$

$$= D_e + d(1 + \cot \theta_{std}) - \frac{P_{std} \cot \theta_{std}}{2} + R_{zero} + all.$$

$$= 12.536 + 1.35 \left(1 + \frac{1}{\sin 30^\circ}\right) - \frac{2}{2 \tan 30^\circ} + 0.05 - 0.00$$

$$R_{min} = 14.904 \text{ mm}$$

$$R_{max} = 15.114 \text{ mm}$$

5

On a T.M.M. the pitch of a screw was measured and the readings (mm) were as follows:

1 st Side	5.02	7.04	9.05	11.03	13.00	14.98	17.02	19.04	21.00	23.03
2 nd Side	6.01	8.03	10.05	12.05	14.02	16.05	18.03	20.05	22.07	24.07

If the screw is a perfect screw in other parameters and it'll fit a nut (perfect nut) of 10 mm height. Will this screw engages fully with the nut or partially; and if partially how many teeth will be engaged? Given that the permissible allowance of the pitch diameter is $\pm 0.06 \text{ mm}$.

$$P_{std} = \frac{H}{\text{no. of threads}} = \frac{10}{10} = 1 \text{ mm}$$

$$P = R_2 - R_1$$

$$P(\text{mm}) \quad 0.99 \quad 0.99 \quad 1 \quad 1.02 \quad 1.02 \quad 1.07 \quad 1.01 \quad 0.99 \quad 1.07 \quad 1.04$$

$$P_{av} = \frac{\sum P_i}{10} = 1.02 \text{ mm}$$

$$\Delta P_i = P_i - P_{std} = P_i - 1$$

$$\Delta P(\text{mm}) \quad -0.01 \quad -0.01 \quad 0 \quad +0.02 \quad +0.02 \quad +0.07 \quad +0.01 \quad -0.01 \quad +0.07 \quad +0.04$$

$$\Delta P_{max} = +0.07 - (-0.01) = 0.08 \text{ mm}$$

$$\Delta P = \Delta P_{max} \cot \theta_{std} = 0.08 \cot 30^\circ = 0.139 \text{ mm (Metric thread)}$$

$$\Delta P > \text{Allowance} \Rightarrow \text{Nut will fit partially}$$

$$\text{Tooth no.} \quad 1 \quad 2 \quad 3 \quad 4 \quad 5 \quad 6$$

$$\Delta P_{accumulated}(\text{mm}) \quad 0.01 \quad 0.02 \quad 0.03 \quad 0.04 \quad 0.06 \quad 0.13$$

5 tooth only fit from nut